

## ASHWORTH ENGINEERING

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## PRODUCT TECHNICAL BULLETIN

## PosiDrive Grid 150 (PDG150)

Patented: US 9,884,723, US 10,364,101, and other US and International patents pending

PosiDrive Spiral<sup>TM</sup> is an alternative to Lotension drive systems. PosiDrive Spiral<sup>TM</sup> features a stainless steel grid belt with driving tabs on the inside edge. The tabs engage and are driven by vertical ribs on the spiral center drum. PosiDrive Spiral<sup>TM</sup> is recommended particularly for applications where product movement is a problem, or where oils or other product characteristics cause operational issues with Lotension drive systems. The PosiDrive Spiral<sup>TM</sup> system is suitable for new or existing spiral conveyors.

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## **POSIDRIVE SPIRALTM OPERATION**

PosiDrive Spiral™ differs from Lotension drive systems in several important ways:

### LINKS WITH DRIVING TABS

PosiDrive Spiral<sup>TM</sup> belt links feature a driving tab on the inside edge of the belt. The tab provides a driving surface to positively engage the drum (see image at right).

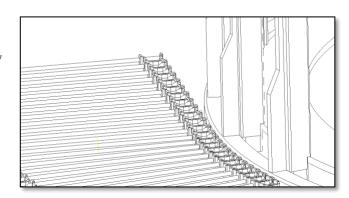
## DRIVING RIBS ON THE CAGE

The PosiDrive Spiral<sup>TM</sup> system features drive ribs on the drum, typically as part of the cage bar cap. The driving rib engages the driving tab of the link to drive the belt in the same way that a sprocket drives a chain. The driving rib projects outward to create a larger drum diameter in the first tier of the spiral and tapers to create a smaller drum diameter above the first tier. The decreasing diameter serves to relieve tension in the belt.



## **GUIDE RINGS**

Guide rings will be placed on the drum where the belt enters and exits. The guide rings allow the links to collapse smoothly before engaging the ribs, then expand smoothly after disengaging from the ribs. Systems that have a belt return on the cage **MUST** also have a return guide ring. Mounting locations for the rings will be provided on the cage bar caps (see image at right).



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#### POSITIVE DRIVE (NO OVERDRIVE)

PosiDrive Spiral<sup>TM</sup> does not rely on friction to drive the belt, as in Lotension drive spirals. Because of the positive engagement between the belt and the drum, overdrive (faster movement of the drum relative to the belt) is eliminated. Belt speed will always match the drum speed precisely. This ensures smooth, constant belt speed without surging, which minimizes movement and damage of delicate products.

#### LUBRICATION

Since the PosiDrive Spiral<sup>TM</sup> system does not rely on friction to drive the belt, it is unaffected by product oils and other lubricants on the drum. The system can be freely lubricated with any suitable lubricant.

### CONSTANT TENSION OUTFEED

Lotension spirals can experience high tensions and even belt flip-ups in the last tier. This occurs whenever oils, frost, or other issues prevent the drum from adequately driving the belt. The outfeed motor is forced to pull with increasing tension in order to maintain the pre-set speed. However, the PosiDrive Spiral<sup>TM</sup> system eliminates this problem. It uses a special outfeed motor programmed to pull the belt at a low, pre-set tension. Belt speed is determined by setting the drum speed. The outfeed motor will automatically match the belt speed without increasing the belt tension.

## **DEFINING CHARACTERISTICS**

**Minimum Turn Ratio:** As low as 1.7:1 standard. *Turn ratios and configurations must be evaluated by* 

Ashworth Engineering.

**Turn Capability:** Turns either clockwise or counterclockwise per system. Turn direction must be

specified at time of order.

**Mode of Turning:** Inside edge collapses in turn.

Width Limits: 12 inch [305 mm] through 54 in. [1372 mm]; wider belt widths need to be reviewed

by Ashworth Engineering.

**Max Allowable Tension:** 400 lbs. [181 kg] through a turn and 800 lbs. [363 kg] in straight run applications

**Longitudinal Pitch:** 1.50 inch [38.1 mm]

**Link Strip Size:** .590 inch x .130 inch [15 mm x 3.3 mm]

**Rod Diameter:** .236 inch [6 mm] **Material:** Stainless steel

**Method of Drive:** Sprocket driven on links at conveyor outfeed, cage driven at belt edge throughout

spiral, and optionally sprocket driven at conveyor infeed

Conveying Surface: 3-5/32 inch [80 mm] less than nominal width

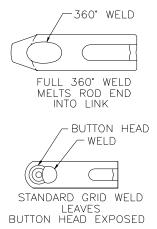
Mesh Overlay: Standard mesh configurations available (see page 4)

## **Improved Weld**

The traditional welded construction of Grid belts fails when the weld breaks. Failure of either the inner or the outer weld allows the link to flex inward when subjected to cyclic loading. The flexing of the link causes fatigue failure at the corners of the link.

Some manufacturers have attempted to slow this process down by including additional welds. However, the weakest weld remains on the inside, the size of which is limited due to the rod size. Too large a weld on the inside will cause the rod to bend when the weld cools, which leads to collapse, tracking and tenting problems.

The Ashworth solution is to create a full 360° weld on the outside edge of the link. This prevents stress on the weld during operation, even with heavier loads. The design and heavier gage of material used for the PosiDrive Spiral™ Omni-Grid 360 Weld links eliminate the need for a weld on the inside of the link. By forming the 360° weld, only on the outside of the link, the inside weld is not necessary, so the belt will not experience the problem of rod bending caused by excessive inside welds.



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#### **Wear Resistant Feature**

The next mode of failure, once weld and fatigue have been eliminated, is belt elongation due to link face wear. The patented wear resistant feature in the link face, included in the PosiDrive Spiral™ Omni-Grid 360 Weld belt, now becomes more important than ever. It provides increased bearing surface to reduce belt elongation.

Patented 360° Weld



Patented Wear Resistant Feature



## **BELT SPECIFICATIONS**

## **MESH OVERLAY:**

### **Designation:**

B X-Y-Z and U X-Y-Z

**First Digit:** B = Balanced Weave; U = Unilateral Weave

X: First Number: No. of Loops per Foot of WidthY: Second Number(s): No. of Spirals per Foot of Length

(8 for 1.5 in. pitch)

**Z:** Third Number: Wire gauge of overlay

## **Examples:**

B30-8-17 U42-8-16

## **OMNI-TOUGH®:**

- Provides a flatter mesh surface with a high resilience to impact.
- Not available in all mesh configurations or for all belt widths.
- Available in 16 ga. (.062 inch [1.6 mm]) and 17 ga. (.054 inch [1.4 mm]).

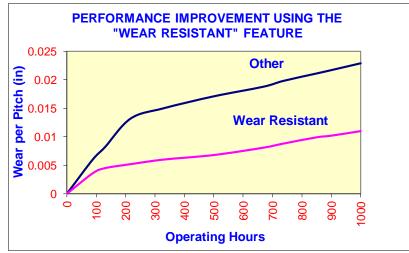
Wire Sizes: 16 ga. [1.6 mm] and 17 ga [1.4 mm].

Material: Stainless Steel high tensile spring wire (Omni-Tough®)

## PATENTED "WEAR RESISTANT" FEATURE

- Standard on all tension bearing links.
- Increases belt life by reducing belt elongation.





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## **BELT WEIGHT**

## **BELT WEIGHT TABLE (WITHOUT MESH)**

PosiDrive Omni-Grid 360 Weld 150 Belts (1.5" nominal Pitch) OA Belt Width Base Belt Weight								
12	305	2.17	3.24					
14	356	2.37	3.53					
16	406	2.57	3.83					
18	457	2.77	4.13					
20	508	2.97	4.43					
22	559	3.17	4.73					
24	610	3.37	5.03					
26	660	3.57	5.32					
28	711	3.77	5.62					
30	762	3.97	5.92					
32	813	4.17	6.22					
34	864	4.37	6.52					
36	914	4.57	6.82					
38	965	4.78	7.13					
40	1016	4.98	7.43					
42**	1067	5.18	7.72					
44**	1118	5.38	8.02					
46**	1168	5.58	8.32					
48**	1219	5.78	8.62					
50**	1067	5.98	8.92					
52**	1118	6.18	9.22					
54**	1168	6.38	9.51					

<sup>\*\*</sup>Engineering review recommended

## MESH WEIGHT TABLE

Mesh Lateral	16	ga.	17 ga.			
Count	lb/ft <sup>2</sup>	kg/m²	lb/ft²	kg/m²		
18	0.53	2.59				
24	0.69	3.38				
30	0.86	4.21				
36	1.03	5.04	0.78	3.82		
42	1.20	5.87	0.91	4.45		
48	1.37	6.70	1.03	5.04		
54	1.54	7.53	1.16	5.67		

## **BELT OPTIONS**

# VARIABLE LOOP COUNT OVERLAY (PATENTED)

Overlay which has varied loop spacing across the width of the belt so that the loops get progressively closer together as the spiral goes from the inside of the belt to the outside of the belt (inside and outside are with respect to a turn).



- Variable Loop Count Overlay is available in 16-gage and 17-gage spring wire.
- The tightest mesh available is a B42 or a U54 at the outside edge. This can progress down to a B18 or a U36 at the inside edge.
- Direction of turn must be specified on the manufacturing order.
- Mesh will be designated, i.e., B42/36-8-17 (balanced 42 mesh spacing outside edge progressing to 36 mesh spacing inside edge); or U48/36-8-16 (unilateral 48 mesh spacing outside edge progressing to 36 mesh spacing inside edge).
- Variable loop counts must be evaluated by Ashworth Engineering.

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#### SPECIAL SPIRALS (PATENTED)

- Available in Omni-Tough® only.
- Available in 16 ga. and 17 ga. only.
- One or more spirals on conveying surface are raised.
- Used as guard edges, lane dividers and flights.
- Maximum height 1 inch [25.4 mm].
- Available Options: height, spacing, location, shape, and number of lanes in belt.



Isosceles Triangle

#### **GUARD EDGES**

## **Integral Guard Edge Link**

- These links have a raised top on the leg adjacent to the conveying surface; the raised top, or guard edge, prevents product slide from the conveying surface.
- Upper edge and lower edge retain #1 round edge for smooth travel over plastic wear strip.
- Offers improved cleanup and sanitation over guard edge plates.
- Available in 1 inch [25 mm] height above the belt surface.
- Available on outside edge only.
- Longitudinal spacing (every pitch, every other pitch, etc.) must be specified.
- Direction of turn (clockwise or counterclockwise) must be specified.



Belt with Integral Guard Edge links

## **SPROCKETS**

#### **UHMW-PE SPROCKETS**

No. of	Ove	rall	Pitch		H	Hub Hub		Bore				
Teeth	Dian	neter	Diameter		Width Diameter		Minimum		Maximum*			
	inch	mm	inch	mm	inch	mm	inch	mm	inch	mm	inch	mm
17	8.50	216.0	8.16	207.3	2.00	51.0	7.43	188.7	0.75	19.05	3.00	76.2

## **NOTES:**

- UHMWPE material type components have a 150°F [66°C] maximum operating temperature.
- Maximum bore sizes listed for UHMWPE material is based on 1/2 inch [12.7 mm] of material above keyway.

#### FILLER ROLLS

• 7-5/8 inch [193.7 mm] diameter filler rolls are recommended with #8-17 tooth sprockets

## SYSTEM REQUIREMENTS

#### **CAGE BAR SPACING:**

Number and spacing of cage bars is dependent on cage diameter vs. belt width (turn ratio).

### **CAGE BAR CAPS:**

PosiDrive Spiral<sup>TM</sup> cage bar caps must be installed on every cage bar. Only PosiDrive Spiral<sup>TM</sup> cage bar caps are designed specifically to work with PosiDrive Spiral<sup>TM</sup> belt and are custom engineered for each spiral system's requirements. PosiDrive Spiral<sup>TM</sup> belt will not work with standard Lotension cage bar caps.

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THEORETICAL BELT PATH

ACTUAL BELT PATH

-SWING WIDE

#### **OUTFEED DRIVE:**

PosiDrive Spiral<sup>TM</sup> systems must be equipped with a special outfeed drive motor that is programmed to operate at a constant torque. Since PosiDrive Spiral<sup>TM</sup> systems operate without overdrive, belt speed will be set by the drum speed. Therefore, the outfeed motor must have the ability to operate at variable speeds to conform to the drum speed. Standard outfeed motors, which operate at one constant speed, will cause operational issues.

#### **INFEED DRIVE:**

In systems with extremely high tensions, an additional motor may be required at the infeed to reduce belt tension entering the spiral. Contact Ashworth Engineering to receive a recommendation.

#### **TERMINALS:**

At all terminals with a wrap of  $120^{\circ}$  or more, the belt should be supported by 4 inch [100 mm] minimum diameter rollers or flanged idlers.

#### **SWING WIDE**

Any belt tends to "swing wide" as it enters or exits the spiral cage, following a path that is offset, but parallel to the normal tangent line to the cage. This phenomenon itself does no damage, but often the belt edge contacts framework that does not leave sufficient clearance for this to occur. The usual reaction of builders or users is to restrict the path of the belt from swinging wide, typically by use of rollers or shoe guides.

However, restraining the belt path can have several adverse effects on belt life:

- The belt can wear through a shoe guide, allowing the edge to snag. This will eventually cause an increase in belt tension and damage the belt edge.
- If the belt is pushed into a straight tangent path, the tension carried in the outside edge of the belt is shifted to the inside edge of the belt, resulting in a pronounced tendency for one edge of the belt to lead the other.

Ashworth recommends that a minimum swing wide clearance of 1 inch per foot [75 mm per meter] of width be built into all conveyors where the belt enters or exits a turn.

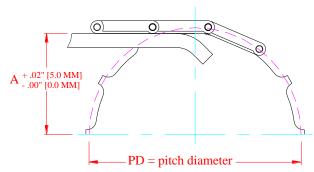


As a rule, support rails are required, with maximum spacings of 18 inches [457.2 mm] on the load side and 24 inches [609.6 mm] on the return side. Rollers may also be used. For light loads, support rails may be placed farther apart – consult Ashworth Engineering for your particular application.

#### Rail Height (with wear strip):

$$A = \frac{1}{2} X PD - 0.25 inch [6.4 mm]$$

This is only a guideline; it does not take into account the influence of speed. At speeds above 75 ft/min [23 m/min], Ashworth recommends increasing the distance A and shortening the wear strips as much as one belt pitch in length. (Nominal Belt Pitch = 1.5 inches [38.1 mm])



## Inner Rail Spacing (with wear strip):

In the first tier, a minimum of 2 ½ inches [63.5 mm] clearance is required between the cage bars (without caps) and the inner support rail, with wear strip installed. The remaining tiers should have a minimum of 1 ¾ inches [44.5 mm] clearance. In existing systems, the inner rail on the first tier may need to be moved.

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## **ENGINEERING CALCULATIONS**

**Turn Ratio:** (for calculating maximum acceptable belt turn ratio for existing systems)

$$TR = ITR \div BW$$
 where  $ITR = Inside Turn Radius$   
 $BW = Belt Width$ 

Turn Ratio is dimensionless. Inside Turn Radius and Belt Width must both be in same unit of measurement, either both in units of inches or both in units of millimeters.

Inside Turn Radius: (for calculating radius dimensions for new system design)

$$ITR = TR \times BW$$
 where  $TR = Turn Ratio$   
 $BW = Belt Width$ 

## **Belt Weight:**

(Weight of Base Belt) + (Weight of Mesh Overlay)

#### Steps of Calculation:

- Determine weight of Base Belt in lb/foot or kg/meter (see chart on page 4).
- Calculate Conveying Surface and convert to units of feet or meters.
- (Conveying Surface = Belt Width 3.14 inch [80 mm])
- This will equal sq. feet [sq. meter] of mesh/foot [meter] of belt length.
- Use the Conveying Surface and Mesh Type to determine weight of mesh in lb/foot or kg/meter (see chart on page 4).
- Add Weight of Base Belt to Weight of Mesh Overlay, lb/foot or kg/meter.
- Multiply calculated value by belt length (feet or meter) for total belt weight in units of lb or kg.

Belt Tension: Needs to be calculated/reviewed by Ashworth Engineering for each application.

## <u>PRODUCT LOADING REQUIREMENTS</u>

All PosiDrive Spiral<sup>™</sup> belts accommodate a turn by collapsing along the inside edge. Product loading must be adjusted accordingly. The allowable loading per length of belt is determined by the ratio of the inside turn radius and the radius to the tension link.

## STANDARD LOADING RECOMMENDATIONS

Allowable Loading per length of belt = (Radius to Tension Link) ÷ (Inside Turn Radius)

#### Sample Calculation:

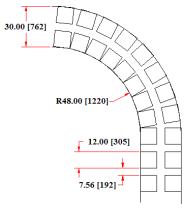
Let BW = Belt Width = 30 inch [762 mm] Let IR = Inside Turn Radius = 48 inch [1219 mm]

Radius to Tension Link = BW + IR

= 30 inch [762 mm] + 48 inch [1219 mm] = 78 inch [1981 mm]

Allowable Loading = 78/48 = 1.63

Which means a minimum space of 63% of the product length is required between products.



Product along inside edge moves closer together; no effect is observed on the product along outside edge. Loading: 1 in 1.63 product lengths.

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## <u>MAINTENANCE</u>

Lower belt tension will improve belt life and reliability. Therefore, proper operation and regular cleaning, lubrication, and maintenance are encouraged. Generally, belt tension increases when product debris (flour, glazing, marinade, etc.) increases the friction between the belt and the support rails. Other factors, such as worn or misaligned components and increased product loading, will also increase belt tension. The following guidelines will help lower belt tension and reduce wear.

#### **CLEANING**

- Clean product debris from support rails.
- Clean ice and product debris from belt, sprockets, and filler rolls to prevent belt damage.
- Observe the effect of temperature on the coefficient of friction between the supports and the belt. Products may leave a slick residue at room temperature that turns into a tar-like substance as temperature decreases. At freezing temperatures, the debris may become slick again or leave a rough surface, depending upon its consistency.

#### LUBRICATION

- Lubricate support rails to reduce friction between rails and belt.
- Lubricate inside edge of the belt (note that this is opposite from general practice with Lotension spirals).
- Lubricate links.

#### OPERATION

- Replace worn wear strips on support rails.
- Remove weight from take-up. Use minimum weight necessary to maintain take-up loop.
- Align sprockets properly and ensure that they do not walk on the shaft.
- Load belt so that belt weight, product loading, friction factors, and belt path do not cause belt tension to exceed
  maximum allowable limit.
- Decrease belt speed.
- Properly adjust discharge helper drive to maintain lowest possible "take-away" tension.

Reference: Product Technical Bulletin "Conveyor Design Guidelines."

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